

Flat Heat Sink, 009'(2.3mm) Thick - Hcrz. Set-up Fig. 3C - Vert Set-up Fig 3D
Figure 5. R_θ Board-Ambient Per Emitter in Free Convection Vs. Foot Print Area

In the vertical orientation, the thermal resistance decreases noticeably as the exposed surface area doubles. The total surface area of the horizontal heat sink equals the foot print area. For the vertical heat sink, the total surface area is double the foot print area. The vertical heat sink is also more efficient due to the nature of free convection. Warm air moving over a vertical surface is more efficient than air that moves vertically away from a horizontal surface. As the foot print areas approach 9in, the R_θ B-A of the two orientations begin to converge. This indicates that as foot print areas approach 9in per emitter, heat sink orientation is not influential. Also, with areas greater than 9in per emitter, there are diminishing reductions in the R_θ B-A. The lower limit for R_θ B-A with increasing area will approach about 10 to 11 ~ C/W.

4. Range of Efficiency with Flat Heat Sinks

The two conditions shown in Figure 5 represent the most efficient (vertical, 2 convective surfaces) and least efficient (horizontal on low conducting insulating foam) configurations of a flat heat sink. Most applications probably fall some where in between. When selecting a heat sink for your application, you will need to determine the most comparable condition. You will also need to assess other factors that might make the R_θ B-A of the larger or smaller than the extremes shown in Figure 5. Mounting the heat sink to a conductive surface or at a 45~ angle, for example, are both factors that would reduce the R_θ B-A compared to the horizontal orientation in Figure 5.

5. Finned (Fig. 3A) vs. Flat Heat Sinks (Fig. 3C) in Free (Natural) Convection

We tested two finned heat sinks with identical 2 in foot print areas, but different exposed surface areas. Increasing the number and length of fins on the heat sink increases the surface area. The fin spacing was about 0.25 in. Figure 6 shows R_θ B-A per exposed surface area for finned heat sinks and flat heat sinks. The heat sinks plotted in Figure 6 are horizontal (Set-up Figure 3A for finned, Figure 3C for flat). The finned heat sinks required more exposed surface area for a given R_θ B-A compared to the flat heat sinks. This shows that a flat heat sink can be effective in thermally managing WAH WANG Power Light Sources with 25 mm emitter spacing. In order to fully utilize the surface area on the finned heat sinks, the fins must lie in parallel with the convection airflow. The finned heat sinks would probably have a slightly lower R_θ B-A if oriented vertically (Set-up Figure 3B).

6. Finned Heat Sinks Reduce Foot Print Size

The Figure 7 shows R_θ B-A per foot print area for finned heat sinks and flat heat sinks. Each of the finned heat sinks had 2 in footprints. The flat heat sinks have footprints equal to the exposed area. A flat heat sink needs about 6 in footprint to match the R_θ B-A of a 2 in footprint finned heat sink. If footprint size is a major design constraint, a finned heat sink offers an efficient solution. The lower limit for R_θ B-A using a 2 in footprint finned heat sink is about 10 to 11 ~ C/W. A heat sink typical of this performance is an AAVID heat sink extrusion part # 65245. A 1.6 in length of this heat sink extrusion has 25 in total surface area with a 2 in footprint. R_θ B-A for this heat sink is plotted in Figure 7. Looking at Figure 5, a 9 in vertical flat heat sink (18 in total surface area) would have about the same R_θ B-A.

B. Heat Sinks in Free Convection - Dense Emitter Spacing When WAH WANG Power LED is densely packed, they function as a single heat source. This chart will help you characterize the WAH WANG Power LED as well as custom Level 2 Boards with emitter spacing between 9 and 13 mm. This chart can also be used to characterize heat sinks for clustered Power LED, with spacing up to about 19 mm. For wider spacing, use the charts in Section B. The following chart in Figure 9 shows the Total Array R_θ B-A vs. heat sink area required for the total array. It is the total array R_θ B-A shown in Figure 2, which is the thermal resistance model for multiple emitter products.

We characterized three types of heat sinks using 12 and 18 WAH WANG power LED Floods. The results are shown in Figure 9. All heat sinks were vertically orientated with free convection on all sides. We tested

both flat plate (see Figure 3D test set-up) and finned heat sinks (see Figure 3B.) Figure 9 should be most useful in sizing heat sinks for custom applications that use ten to twenty Power LED. However, it can also be used as a rough guide for sizing heat sinks for applications with about 3 to 20 densely spaced Power LED.

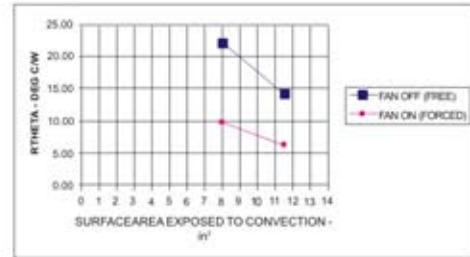


Figure 8. R_θ Board-Ambient per Emitter - Free Conv. (Test Set-up Fig. 3A) vs. Forced Conv. (Test Set-up Fig. 3E) - 42f/min (12.8m/min) Air Flow with Fan On

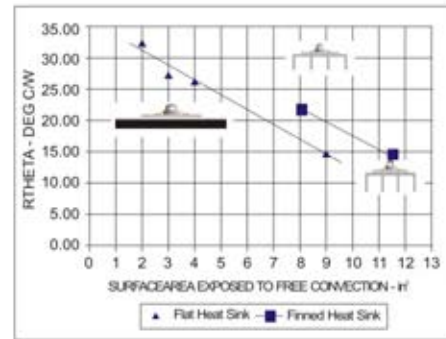


Figure 6.

R_θ Board-Ambient per Emitter in Free Conv. Horizontal Flat Heat Sink - Set-up Fig. 3A vs. Horizontal Finned Heat Sink- Set-up Fig. 3C

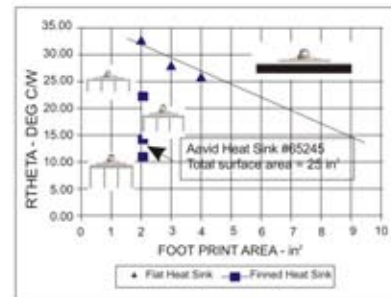


Figure 7.

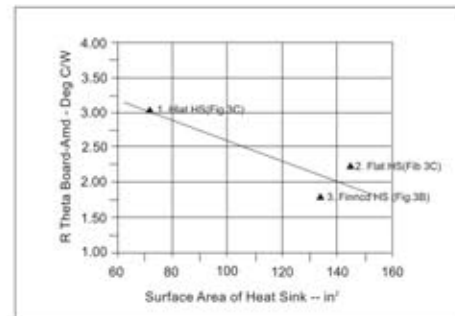


Figure 9. High Density Emitter Heat Sink Total Array Thermal Resistance (Board to Ambient) vs. Surface Area Exposed